

**IN THE SPECIFICATION:**

**Please amend paragraph [0001] as follows:**

[0001] This application is a continuation of U.S. patent application Ser. No. 10/374,026 filed on Feb. 25, 2003, which is a continuation of U.S. application Ser. No. 09/469,549 filed Dec. 22, 1999, now U.S. Patent no. 6,629,708, which is continuation-in-part of U.S. patent application Ser. No. 09/054,186 filed on Apr. 2, 1998, now U.S. Pat. No. 6,131,963, which is a continuation-in-part of U.S. patent application Ser. No. 08/834,255 filed on Apr. 15, 1997, now U.S. Pat. No. 5,882,050, the entire disclosures of which are fully incorporated herein by reference.

**Please amend paragraph [0044] as follows:**

[0044] Referring now to the drawings wherein the showings are for the purposes of illustrating preferred embodiments of the invention only and not for purposes of limiting same, ~~FIGS. FIG. 2~~ illustrate the overall arrangement of a fitting incorporating the invention. It should be noted that in many of the illustrations herein of the ferrule profiles, the ferrules are shown in partial cross-section for clarity and ease of understanding, particularly for views of the ferrule geometry and profile wherein it is only necessary to illustrate a portion of the entire ferrule in sectional view. The ~~FIGS. FIG. 2~~ embodiment has the major components identified with the same reference numerals used with respect to the description of the prior art device of FIGS. 1 and 1A. A description of a FIG. 1 element is to be taken as equally applicable to the ~~FIGS. FIG. 2~~ elements that are correspondingly numbered unless otherwise noted. In particular, in the ~~FIGS. FIG. 2~~ embodiment, the rear ferrule 22, has been modified in a manner to cause the reaction forces acting between the front ferrule through the rear ferrule to the nut to have a significant force component that is directed radially outward. This is in contradistinction to the FIGS. 1 and 1A embodiment wherein the force component under consideration has a high axial component. Specifically, as shown in FIG. 4, force component A extends generally axially of the rear ferrule 22 and results in an increase in the loads applied at the radial inner face of the rear ferrule driven surface 28 and nut shoulder 32. As previously discussed, high localized loading or stress concentration in this area produces high torque and galling.

**Please amend paragraph [0048] as follows:**

**[0048]** Another important feature of the invention is best exemplified by comparing the rear ferrule 22 of FIG. 1 with the rear ferrule 22 of the ~~FIGS. 1 and 1A~~ FIG. 2 embodiment. Particularly, the outer radial wall 50 of the rear ferrule 22' includes a conical section that increases in radial dimension as it extends from the forward nose portion 52, that is received in the rear chamfer region 53 of the front ferrule, to the rear flange 26". In the prior art arrangement (FIGS. 1 and 1A), the rear ferrule has a cylindrical through bore and an outer radial wall that extends parallel to the inner surface defining the through bore in this region. In other words, the rear ferrule has a constant annular wall thickness "t". In the embodiment of ~~FIGS. 1 and 1A~~ FIG. 2 the outer wall has the conical or tapered configuration that provides sufficient wall thickness t and controlled deformation of the nose portion when the recess is incorporated into the modified rear ferrule. Preferably, the outer wall 50 has a generally uniform angle or taper as it extends between the reduced dimension of forward nose portion 52 received in the camming mouth of the front ferrule and the enlarged diameter of rear flange 26". Again, this provides controlled deformation of the rear ferrule so that forward nose portion 52 is plastically deformed radially inward along forward contact area 44 into gripping, sealed engagement with the outer wall of the tube. Note that in FIG. 4 the recess 40 is so shaped as to appear that the dimension "t" is constant, though it need not be. For example, if the recess 40 is formed such as in many of the illustrations of FIGS. 14, 17 and 18, the tapered outer wall 50 provides a non-uniform thickness "t" between forward nose portion 52 and rear flange 26'.

**Please amend paragraph [0057] as follows:**

**[0057]** FIGS. 8 and 9 show ~~another~~ a modified rear ferrule in accordance with the teachings of the present invention. This rear ferrule is the same as shown in FIG. 5. As particularly evident in FIG. 9, the region of high stress concentration is substantially reduced in size when compared to FIG. 7. This indicates that the stresses have been more uniformly dispersed over the rear face of the flange of the rear ferrule. Thus, the torque is reduced and the potential for galling is likewise reduced.